

# Preliminary Design for Assembling and Manufacturing Sports Equipment: A Study Case on Aerobic Walker

Zainal Arifin, Singgih D. Prasetyo, and Aditya R. Prabowo

Department of Mechanical Engineering, Universitas Sebelas Maret, Surakarta, Indonesia  
Email: zainal\_arifin@staff.uns.ac.id (Z.A.); singgihdwiprasetyo22@student.uns.ac.id (S.D.P.)  
aditya@ft.uns.ac.id (A.R.P.)

Joung H. Cho

Department of Industrial Design, Pukyong National University, Busan, South Korea  
Email: jhcho7@pknu.ac.kr

**Abstract**—One way to increase the interest of the community in exercising is providing sport equipment which is integrated with the tourist site. One exercise tool that can be applied is aerobic walker type outdoor fitness equipment. Aerobic walker is a tool with the principle of static walking. This article discusses the planning of aerobic walker type outdoor fitness equipment products using the design for manufacturing and assembly (DFMA) method and calcifications using the house of quality (HOQ) methods in which the design is based on drawings, materials, analysis, and equipment production plans. The method used in this research is action research, numerical engineering and application of simulation using the Autodesk Fusion 360 application. The design of the tool can be used by two people with an average weight of 80 kg and operating with a maximum speed of 30 rpm. The design results are divided into 3 parts: the main frame, foot handle, and footrests made of carbon steel with 6010 bearing types. The safety value of the tool for safety factor and the tool bearing age factor are 15 and 12128.52 hours, respectively. Based on consumer needs with the house of quality method, the design of this tool has a value of 3.9 and is better than other fitness equipment (52 hours).

**Index Terms**—preliminary design, assembly and manufacture reference, aerobic walker, DFMA, HOQ

## I. INTRODUCTION

One of the shortcomings of tourism area promoting health awareness is the lack of awareness of people surroundings about exercise. This indicated by the lack of availability of sport facilities in that area. The majority of people are less interested in sports that are carried out in special places as a means of recreation. One of the ways to overcome this problem is by providing sports facilities

that are integrated with tourist parks in order to attract the interest of the community in exercising [1-4].

One of sport facilities that can be used by all groups is outdoor fitness equipment, because it can be applied as a recreation for people who visit the tourist parks. That suitable fitness equipment is called aerobic walkers which have principle of static walking. Therefore, this equipment can encourage people to do physical activities that are easy, simple, and inexpensive. If this can work effectively, this physical activity can increase surrounding community health and reduce the chance to get disease. However, the use of an aerobic walker type outdoor fitness equipment requires proper planning and design so that the equipment can be used for a long time and can be used by everyone. The design is possibly adopted based on engineering design as previously conducted by Caesar et al. [5] and Ary et al. [6] which considered material and structure geometry in designing hardness tester facility and analyzing structural design of eco-vehicle chassis.

Based on these problems, the authors are interested in planning an aerobic walker type outdoor fitness equipment product. Planning is conducted using the design for manufacture and assembly method. The plan is carried out based on drawings, materials, and components needed in the tool-making process. In addition, this investigation is complemented by a quality function deployment (QFD) process using the house of quality (HOQ) method [7-10].

## II. DESIGN METHODOLOGY

The research method used in this research is the Action Research (AR) method. Action research aims to improve the ways of practitioners in applying appropriate fields of science [8]. In addition, this study uses engineering methods based on the design activities. The design was made by using numerical methods and simulation using the Fusion 360 software. In general, the flowchart of this research is shown in Fig. 1.

---

Manuscript received July 8, 2020; revised January 3, 2021.  
Grant name: PNBP UNS Pengabdian Kepada Masyarakat (PkM)  
Grant no.: 453/UN27.21/PN/2020

### A. Design Concept

The design concept is carried out to identify consumer needs for aerobic walker type outdoor fitness equipment. This is intended as a basis for building a tool so that it can be used efficiently and effectively. The design concept in this paper is carried out by active action research with interviewing several communities. The results of the interview are also used to consider the assessment of the house of quality (HOQ) method.

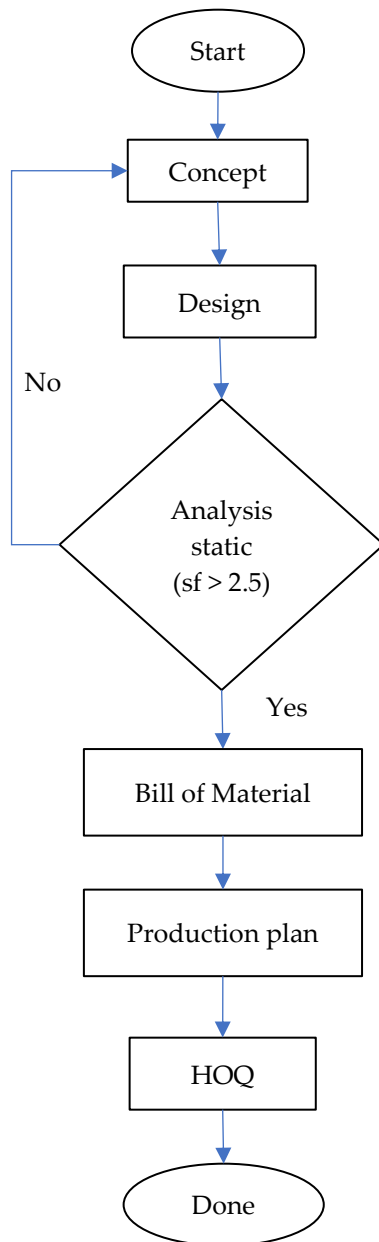


Figure 1. Flow chart of the current design research.

### B. Design

Design is carried out based on the design concept and identification of consumer needs that have been made so that the product's HOQ value can be achieved properly. The sketch is divided into several parts in order to create an effective and efficient product. The most important

part in designing an aerobic walker type outdoor fitness equipment is the bearing in which the choice of bearing is based on the equation listed in Table I. Ashby diagram is used for the selection of materials to get more appropriate design concepts.

TABLE I. FUNDAMENTAL PARAMETERS IN BEARING DESIGN [11]

Type	Formula	Information
Finding the Side Force (W)	$W = Wi \cdot (F_1 \cdot F_2) \quad (1)$	$Wi = 0.5$ $F_1 =$ Attraction style $F_2 =$ Loose side tensile force
Finding the Gravity (Wp)	$Wp = (\pi \cdot D^2 \cdot L) \cdot M \quad (2)$	$D =$ shaft diameter $L =$ shaft length $M =$ density of shaft
Specific Equivalent Radial Load	$Pr = X \cdot V \cdot Fr + Y \cdot Fa \quad (3)$	$V =$ factor which depends on rotating bearing and rotating inner ring $X =$ Radial factor $Y =$ angular axial factor $Fa =$ Axial load $Fr = W + Wp + Wi$
Planned Bearing Life	$H = j \cdot T \cdot h \quad (4)$	$j = 8$ hours $H = 2$ years $h = 365$ days
Selection of bearings (Dynamic load)	$C^3 = \left( \frac{H \cdot 60 \cdot n \cdot Pr^3}{10^6} \right) \cdot Sf \quad (5)$	$C =$ dynamic load (kg) $n = 30$ rpm $Sf = 20$

### C. Design Analysis

The form of the existing design is carried out a static analysis for the aerobic walker type outdoor fitness equipment framework. Analysis was conducted by simulating the load on the framework using the Autodesk Fusion 360 software application. This aims to determine the reliability of the strength and safety factor of the tool. In order to determine the reliability of bearing life, nominal factor analysis and bearing reliability factor were carried out. The analysis was carried out using the numerical method as in Table II.

### D. Bill of Material

Bill of material is an identification in classifying the needs for the type of shape and material of each design part for fabrication. This is the basis for providing a budget for building a design [12]. The prices listed in this paper correspond to market prices as of May 2020 in the city of Surakarta.

### E. House of Quality (HOQ)

House of Quality (HOQ) is a method of Quality Function Deployment to identify the value of the product as desired by consumers. HOQ can be used for initial identification in understanding the shape of the design, characteristics of the sub system, the manufacturing process, and ensuring product quality [13]. In addition, this method can compare the characteristics of critical components and key operations in production.

TABLE II. PARAMETERS FOR NUMERICAL CALCULATION [11]

Type	Formula	Information
Speed Factor (Fn)	$Fn = \left(\frac{33.3}{n}\right)^{\frac{1}{3}}$ (6)	$n = 30$ rpm
Bearing General Factor (Hfn)	$Hfn = Fn \cdot \frac{C}{Pr}$ (7)	$C$ = dynamic load bearing used $Pr$ = design load / actual
Bearing Nominal Factor (LH)	$LH = 500 \cdot Hfn$ (8)	
Bearing Life Reliability Factor (Ln)	$Ln = a_1 \cdot a_2 \cdot a_3 \cdot LH$ (9)	$a_1$ = 96% reliability $a_2$ = material factor $a_3$ = work factor
Safety Bearing Life (Si)	$Si = \left(\frac{Ln}{H}\right)$ (10)	

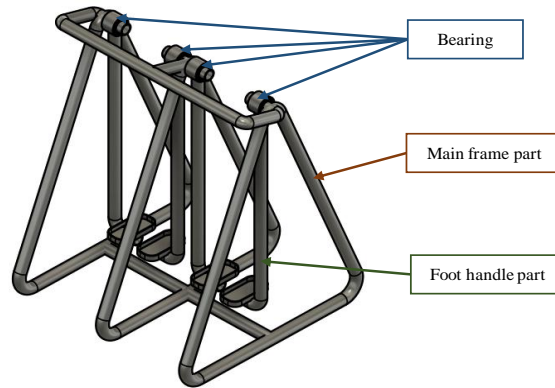


Figure 2. Three-dimensional geometry of the aerobic.

### B. Engineering Drawing for Assembly and Manufacture

After having a basic design, a tool drawing was carried out using the Autodesk Fusion 360 software application. The image was made into 3 parts: main framework, footrests, and handle legs as shown in Fig. 2. The tool size for each part is adjusted to the height and the average load [14]. Thus, the equipment can have good safety standards as shown in Figs. 3 and 4 (the unit is in mm).

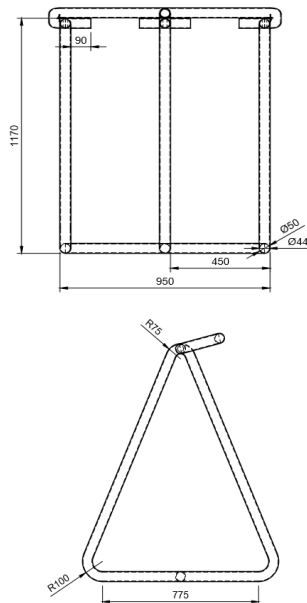


Figure 3. Dimensions specification of the main frame.

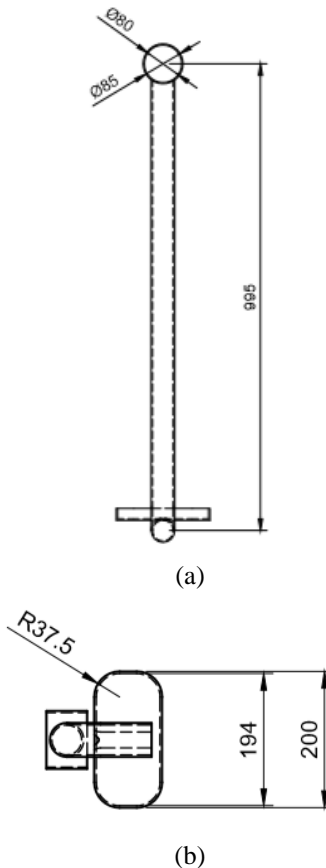


Figure 4. Dimensions specification of design part: (a) foot handle and (b) footrest.

Bearings were located at the joint of main framework and foot handle. Two parts of bearing were prepared in between the shafts for each sub part handle feet. Shaft diameter is designed at 50 mm with a length of 95 mm. Detailed image of the bearing can be seen in Fig. 5.

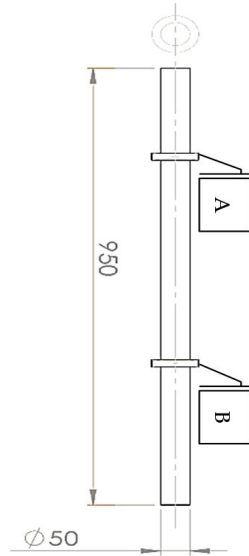


Figure 5. Dimensions specification of the bearing shaft.

The bearing shaft has two supports at point A and point B. Two bearings are the same because the diameter of the supports is the same. The force acting on the bearing is radial force; there is no axial force because no force is exerted on the horizontal axis towards the shaft. Therefore, the bearing type used is Single Row Angular Contact Bearing. The carbon steel shaft can spin up to a maximum of 30 rpm, while the tensile forces on the tight side and loose side are 100 kg and 80 kg. Details of bearing selection design can be seen in Table III.

TABLE III. RESULTS FOR THE BEARING DESIGN

Type	Result
Finding the Side Force ( $W$ )	10 kg
Finding the Gravity ( $Wp$ )	58.57 kg
Specific Equivalent Radial Load	38.67
Planned Bearing Life	5840 hours
Selection of bearings (Dynamic load)	1693.6 kg

From the design calculations and the diameter of the shaft size with the total dynamic load, a bearing 6010 is used with the following specifications which detailed image of the bearing is shown in Fig. 6.

- Diameter ( $d$ ): 50 mm
- Diameter ( $D$ ): 80 mm
- Wide bearing ( $B$ ): 16 mm
- Radius ( $r$ ): 1.5 mm
- Specific dynamic nominal capacity ( $C$ ): 1710 kg
- Specific static nominal capacity ( $Cp$ ): 1430 kg

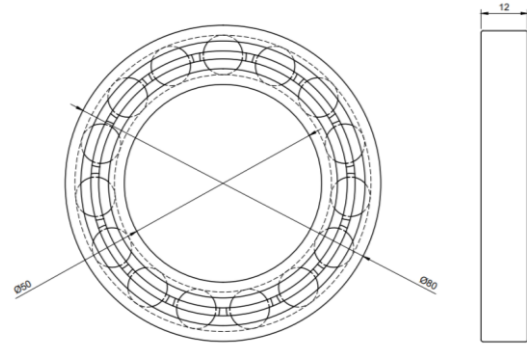
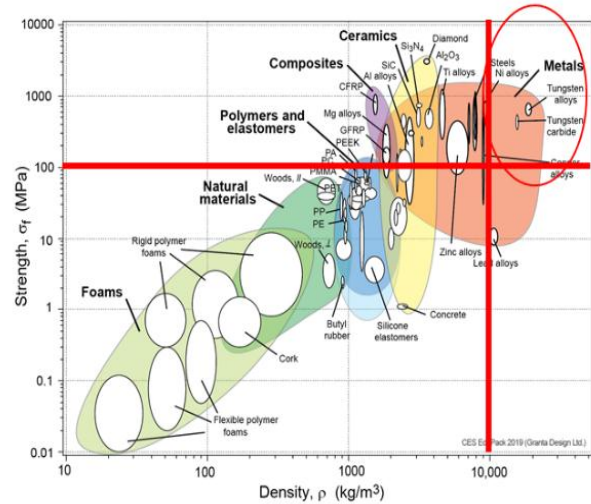
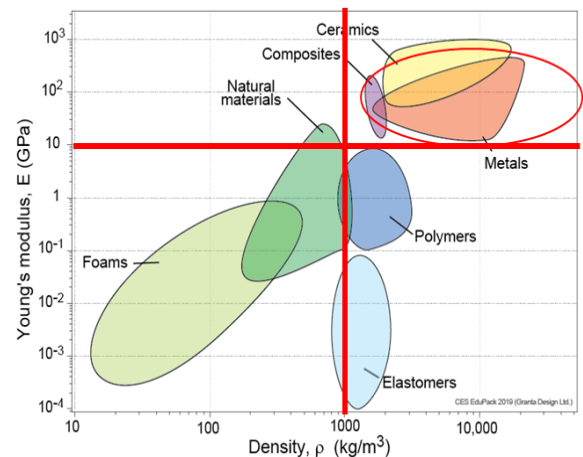


Figure 6. Dimensions specification of the bearing.

Based on the drawings and the design, the basic materials for the tools were selected using the Ashby diagram [15-17]. The selection is based on the number of strength and density which fulfil criteria as shown in the red circle line. In addition, identification of material requirements is also based on classification material young modulus, thermal, and relative coast as shown in Fig. 7. From the selection, materials that match the criteria is in the family of metal so that carbon steel was used in this design.

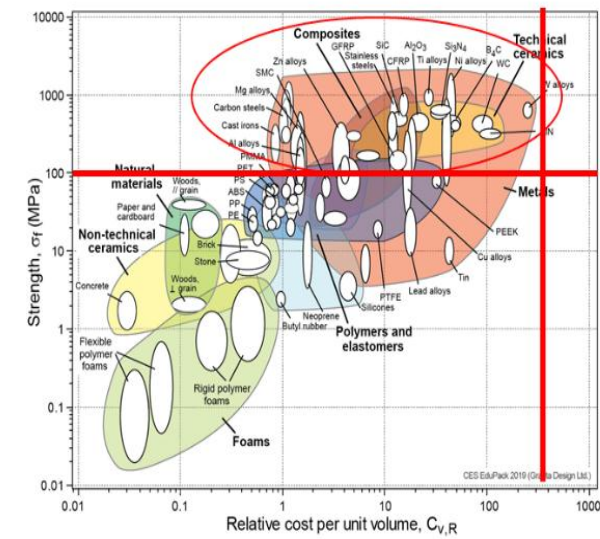


(a)

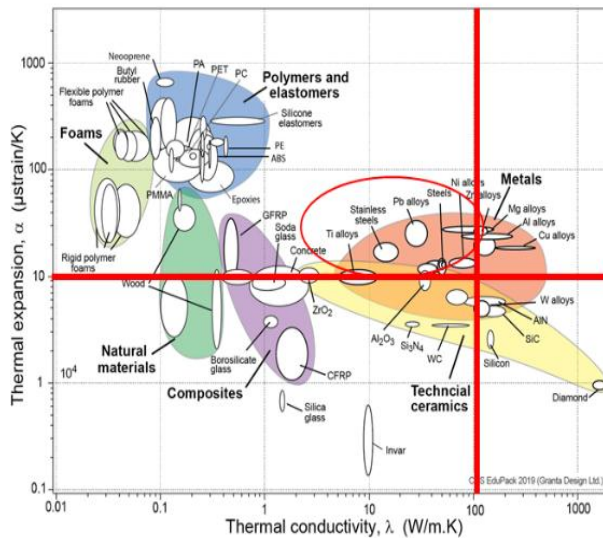


(b)





(c)

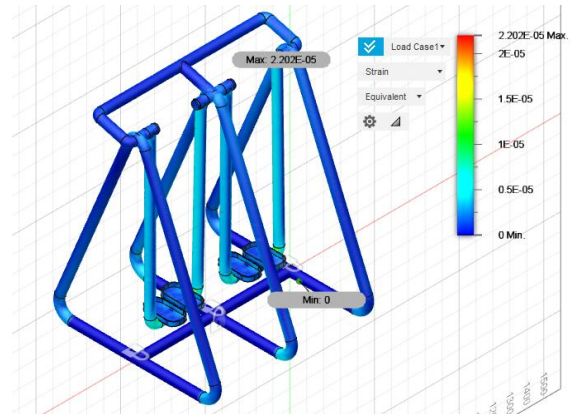


(d)

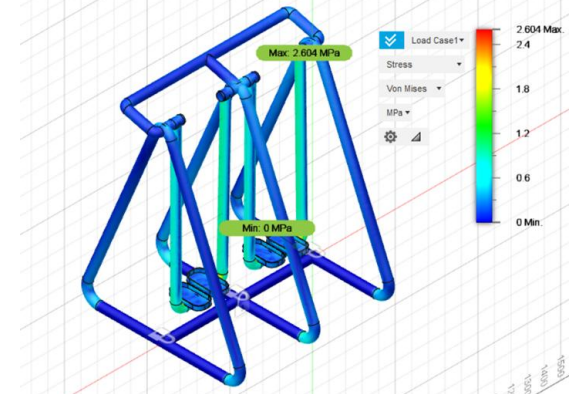
Figure 7. Material selection is based on (a) strength-density, (b) young modulus-density, (c) strength-coast, and (d) thermal characteristic.

### C. Numerical Analysis on the Design

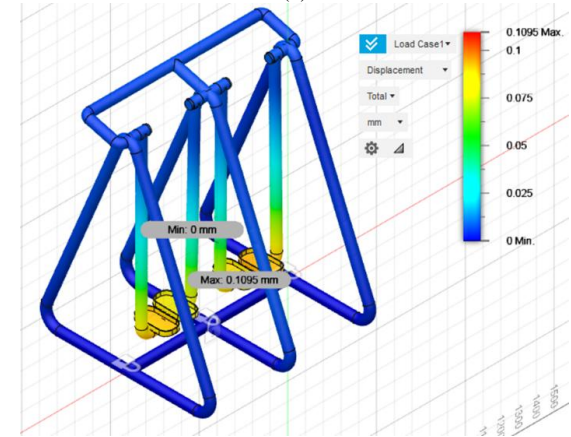
Static analysis on tools outdoor fitness type aerobic walker was performed using the Autodesk Fusion 360 deploying finite element codes in simulation [18-25]. The loading is carried out evenly on the footrests and bearing pipes. In this analysis the pedestal is rigidly at the bottom of the pipe. It can be seen that the design has a maximum value of strain, stress, displacement, and safety factor of 0.000022, 2.604 MPa, 0.1095, and 15 as shown in Fig. 8. This indicates that the materials and the structure of the framework used in the design have good safety standards.



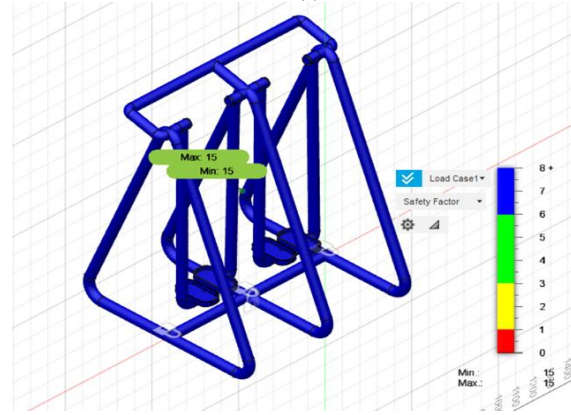
(a)



(b)



(c)



(d)

Figure 8. Simulation results of the structural design: (a) strain pattern, (b) structural stress, (c) component displacement and (d) safety factor.

For the reliability of the bearings, the analysis results show that the bearings are estimated to have a better bearing life factor than the design results. This is because the bearing life factor used has a value twice greater than the desired bearing life, as calculated in Table IV [9]. Therefore, tool can operate with a longer shelf life.

TABLE IV. RELIABILITY ANALYSIS ON THE BEARING DESIGN

Type	Result
Speed Factor (Fn)	1,035
Bearing General Factor (Hfn)	45.76
Bearing Nominal Factor (LH)	22884 hours
Bearing Life Reliability Factor (Ln)	12128.52 hours
Safety Bearing Life (Si)	2

#### D. Bill of Material

After the design analysis produces the desired data, it is then necessary to plan the purchase of the need to fabricate the equipment. This aims to determine the design can be made with existing coast boundaries. It can be seen that fabrication requires a material with a total price of Rp. 8,165,600 as shown in Table V. It means that the design is still within the cost range for fabrication. In addition, this design has a cheaper build value than other fitness equipment because the design does not require other fuels in operation.

TABLE V. BILL OF MATERIAL DESIGN

Name	Unit	Volume	Price (IDR)	Total (IDR)
<b>Main Frame &amp; Foot Handle</b>				
Steel pipe Ø50 mm	Meter	5	200000	1000000
Steel pipe Ø80 mm	Meter	5	450000	2250000
<b>Foot step</b>				
10 mm thick steel plate	Sheet	1	500000	500000
PP plate 3 mm	Sheet	1	300000	300000
<b>Connection</b>				
M10 nut	Seed	50	5000	250000
M10 bolt	Seed	50	5000	250000
Bearings	Seed	8	58200	465600
<b>Support</b>				
Iron paint	Cans	6	150000	900000
RD-260 welding electrode	Pack	1	150000	150000
Cement	Sak	5	150000	750000
Sand	Colt	0.5	400000	200000
Reinforcement Iron	Meter	5	200000	1000000
Gravel	Colt	0.5	300000	150000
Total				8165600

#### E. Production Plan

In production planning on tool design outdoor fitness type aerobic walker divided into two: determining the use of the machine and the treatment of each part [26]. It can be seen that the production planning is divided into 6 parts, where 3 parts are fabricated independently and 3 parts are purchased as shown in Table VI. The production classification can be seen in Table VII.

TABLE VI. PLANNING FOR THE ENGINEERING DESIGN

Part Number	Part name	Tapping	Counter boring	Drilling	Milling	Grinding	Shaping	Fillet	Chamfer	Others
1	Main Framework	N	Y	Y	N	Y	Y	Y	Y	
2	Foot Handle	N	Y	Y	N	Y	Y	Y	Y	
3	Foot step	Y	N	Y	Y	Y	Y	Y	Y	
4	Bearings									Purchased
5	M10 nut									Purchased
6	M10 bolt									Purchased

TABLE VII. CLASSIFICATION OF DESIGN PRODUCTION

Part		DFA Compl. <sup>1</sup>		Func. Analy. <sup>2</sup>		Er. <sup>3</sup>	Handling			Insertion								
No.	Name	a	b	c	d	e	f	g	h	i	j	k	l	m	n	o	p	q
1	Main Framework	1	1	N	H	Y	N	N	N	Y	Y	N	N	Y	Y	Y	Y	Y
2	Foot Handle	2	4	N	H	Y	N	N	Y	Y	Y	Y	Y	Y	Y	Y	Y	Y
3	Foot step	2	4	N	H	Y	N	N	Y	Y	Y	Y	Y	Y	Y	Y	Y	Y
4	Bearings	1	4	Y	M	N	Y	N	N	Y	Y	N	Y	Y	N	N	Y	Y
5	M10 nut	25	25	Y	L	N	Y	N	N	Y	Y	N	N	N	N	N	N	N
6	M10 bolt	25	25	Y	L	N	Y	N	N	Y	Y	N	N	N	N	N	N	N

a. Number of Parts (Np)	j. Holding down required
b. Number of Interfaces (NI)	k. Resistance to Insertion
c. Part can be standardized	l. Obstructed access / visibility
d. Cost (Low / Medium / High)	m. Re-oriented Work Piece
e. Assemble part wrong way around	n. Screw / Drill / Twist / Rivet / Bend / Crimp
f. Tangle / Nest / Stick Together	o. Weld / Solder / Glue
g. Flexible / Fragile / Sharp / Slippery	p. Paint / Lube / Heat / Apply liquid or gas
h. Pliers / Tweezers / Magnifying Glass	q. Test / Measure / Adjust
i. Difficult to align / locate	

Note: <sup>1</sup> Compl is complexity; <sup>2</sup> Funct. Analy. is functional analysis; and <sup>3</sup> Er. is error.

#### F. House of Quality

The quality of product or tool design is evaluated using the house of quality system. Evaluation is to find out that the design product has a better value for consumer needs than other products [27]. The design results show that it has an added value on a strong frame, can be used by more than one person, and does not drain a lot of energy during operation. When compared with competitors or other products, the design has a better value of 3.9 as shown in Fig. 8.

#### G. House of Quality

The quality of product or tool design is evaluated using the house of quality system. Evaluation is to find out that the design product has a better value for consumer needs than other products. The design results show that it has an added value on a strong frame, can be used by more than one person, and does not drain a lot of energy during operation. During comparison with competitors or other products, the design has a better value of 3.9 as shown in Fig. 9 [28-30].

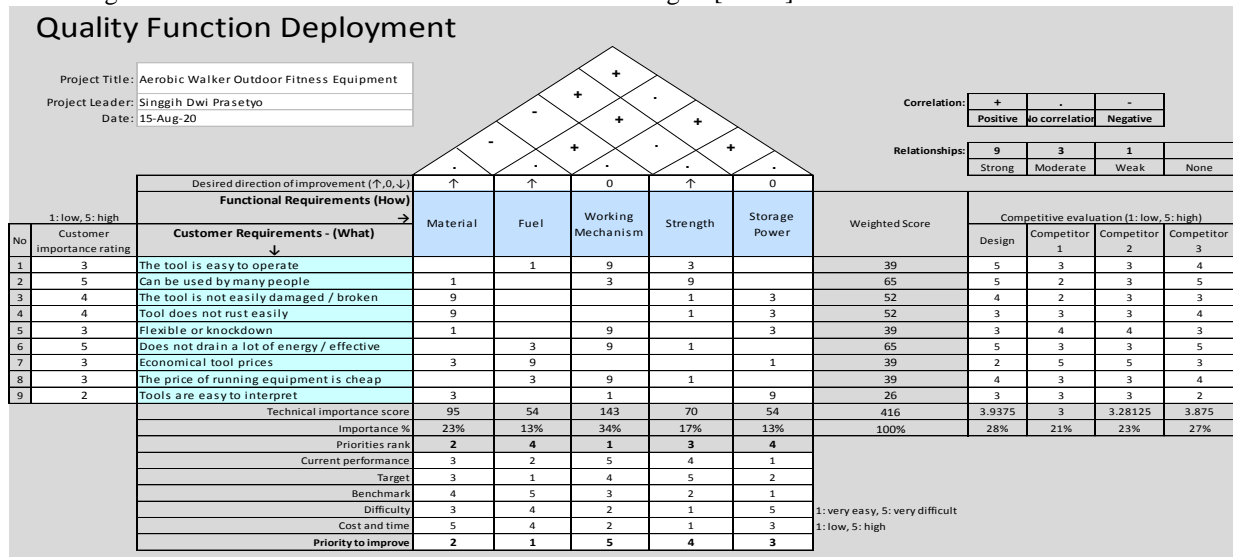


Figure 9. House of Quality (HOQ) for the proposed aerobic equipment design

#### IV. CONCLUSIONS

Based on the house of quality, this tool has the highest product value compared to other types of competitors. From the design that has been made, the authors hope to help the wider community in the process of determining the manufacturing tool.

#### CONFLICT OF INTEREST

The authors declare no conflict of interest.

#### AUTHOR CONTRIBUTIONS

Conceptualization, Z.A. and S.D.P.; methodology, J.H.C.; software, A.R.P.; validation, Z.A., J.H.C. and A.R.P.; formal analysis, S.D.P.; investigation, S.D.P.

The author suggests that readers can do and apply the making of tools according to the design results. The results of the manufacture can be tested at loads that are in accordance with the design basis, trials are carried out to be able to assist in the validity of this design. The design of an aerobic walker type outdoor fitness equipment using the design for manufacture and assembly method has been made. The basic concept of designing a tool for use by two people with an average weight of 80 kg operating at a maximum speed of 30 rpm. The design obtained from the tool is divided into 3 parts: the main frame, foot handle, and footrest. The material used is carbon steel with a bearing type of 6010. The results of tool design analysis have a better safety value determined by the safety factor and the age factor with 15 and 12128.52 hours respectively. The total price needed to make the tool is Rp. 8,165,600 based on the bill of material approach in Indonesia.

Resources, Z.A.; data curation, A.R.P. and J.H.C.; writing—original draft preparation, S.D.P.; writing—review and editing, A.R.P. and Z.A.; visualization, S.D.P.; supervision, J.H.C. and Z.A.; project administration, Z.A.; funding acquisition, Z.A. All authors have read and agreed to the published version of the manuscript.

#### REFERENCES

- [1] D. F. Cronjé and E. D. Plessis, "A review on tourism destination competitiveness," *Journal of Hospitality and Tourism Management*, vol. 45, pp. 256-265, 2020.
- [2] M. Robaina, M. Madaleno, S. Silva, C. Eusébio, M. J. Carneiro, C. Gama, K. Oliveira, M. A. Russo, and A. Monteiro, "The relationship between tourism and air quality in five European countries," *Economic Analysis and Policy*, vol. 67, pp. 261-272, 2020.
- [3] K. Tomej, and Z. Xiang, "Affordances for tourism service design," vol. 85, pp. 103029, 2020.
- [4] P. Remoaldo, J. Serra, N. Marujo, J. Alves, A. Gonçalves, S. Cabeça, and N. Duxbury, "Profiling the participants in creative tourism activities: Case studies from small and medium sized cities and rural areas from Continental Portugal," *Tourism Management Perspectives*, vol. 36, pp. 100746, 2020.
- [5] B. P. P. Caesar, H. Hazimi, H. Sukanto, and A. R. Prabowo, "Development of novel design and frame structural assessment on mitutoyo's auto checking hardness machine using reverse engineering approach: Series hr-522 hardness tester," *Journal of Engineering Science and Technology*, vol. 15, no. 2, pp. 1296 - 1318, 2020.
- [6] A. K. Ary, A. R. Prabowo, and F. Imaduddin, "Structural assessment of alternative urban vehicle chassis subjected to loading and internal parameters using finite element analysis," *Journal of Engineering Science and Technology*, vol. 15, no. 3, pp. 1999 - 2022, 2020.
- [7] L. Ocampo, J. J. T. Labrador, A. M. B. Jumao-as, and A. M. O. Rama, "Integrated multiphase sustainable product design with a hybrid quality function deployment – multi-attribute decision-making (QFD-MADM) framework," *Sustainable Production and Consumption*, vol. 24, pp. 62-78, 2020.
- [8] H. Wang, Z. Fang, D. Wang, and S. Liu, "An integrated fuzzy QFD and grey decision-making approach for supply chain collaborative quality design of large complex products," *Computers & Industrial Engineering*, vol. 140, pp. 106212, 2020.
- [9] A. Chen, M. Dinar, T. Gruenewald, M. Wang, J. Rosca, and T. R. Kurfess, "Manufacturing apps and the dynamic house of quality: towards an industrial revolution," *Manufacturing Letters*, vol. 13, pp. 25-29, 2017.
- [10] J. Village, F. A. Salustri, and W. P. Neumann, "Using action research to develop human factors approaches to improve assembly quality during early design and ramp-up of an assembly line," *International Journal of Industrial Ergonomics*, vol. 61, pp. 107-119, 2017.
- [11] W. S. Widodo, H. Istiqbaliah, "Planning of shallot slicer machine with a vertical slicer (shallot slicer) with a capacity of 1 kg/minute," *Nusantara of Engineering*, vol. 2, no. 1, pp. 30-36 (in Indonesian).
- [12] B. Agard, and B. Penz, "A simulated annealing method based on a clustering approach to determine bills of materials for a large product family," *International Journal of Production Economics*, vol. 117, no. 2, pp. 389-401, 2009.
- [13] T. Park, and K. J. Kim, "Determination of an optimal set of design requirements using house of quality," *Journal of Operations Management*, vol. 16, no. 5, pp. 569-581, 1998.
- [14] D. Bertol, "Designing and making a movement infrastructure," *Procedia Technology*, vol. 20, pp. 72-78, 2015.
- [15] Z. Mahmood, I. Haneef, and F. Udrea, "aterial selection for Micro-Electro-Mechanical-Systems (MEMS) using Ashby's approach," *Materials & Design*, vol. 157, pp. 412-430, 2017.
- [16] A. Rashedi, I. Sridhar, and K. J. Tseng, "Multi-objective material selection for wind turbine blade and tower: Ashby's approach," *Materials & Design*, vol. 37, pp. 521-532, 2012.
- [17] D. U. Shah, "Natural fibre composites: Comprehensive Ashby-type materials selection charts," *Materials & Design (1980-2015)*, vol. 62, pp. 21-31, 2014.
- [18] A. R. Prabowo, F. B. Laksono, and J. M. Sohn, "Investigation of structural performance subjected to impact loading using finite element approach: case of ship-container collision," *Curved and Layered Structures*, vol. 7, pp. 17-28, 2020.
- [19] Ikhsan, J. Triyono, A. R. Prabowo, and J. M. Sohn, "Investigation of Meshing Strategy on Mechanical Behaviour of Hip Stem Implant Design Using FEA," *Open Engineering*, vol. 10, pp. 769-775, 2020.
- [20] A. S. Dabit, A. E. Lianto, S. A. Branta, F. B. Laksono, A. R. Prabowo, and N. Muhyat, "Finite Element Analysis (FEA) on autonomous unmanned surface vehicle feeder boat subjected to static loads," *Procedia Structural Integrity*, vol. 27, pp. 163-170, 2020.
- [21] A. R. Prabowo, T. Muttaqie, J. M. Sohn, and D. M. Bae, "Nonlinear analysis of inter-island RoRo under impact: effects of selected collision's parameters on the crashworthy double-side structures," *Journal of the Brazilian Society of Mechanical Sciences and Engineering*, vol. 40, pp. 248, 2018.
- [22] A. R. Prabowo, A. Bahatmaka, J. H. Cho, J. M. Sohn, D. M. Bae, S. Samuel and B. Cao, "Analysis of structural crashworthiness on a non-ice class tanker during stranding accounting for the sailing routes," *Maritime Transportation and Harvesting of Sea Resources*, vol. 1, pp. 645-654, 2016.
- [23] A. R. Prabowo, J. M. Sohn, D. M. Bae, and J. H. Cho, "Estimating structure response and progressive failure of a ship hull under side-bow collisions," *Tehnicki Vjesnik*, vol. 25, no. 5, pp. 1513-1522, 2018.
- [24] A. Nugroho, H. Nubli, A. R. Prabowo, and H. Yudo, "Finite element based analysis of steering construction system of orca class fisheries inspection ship," *Procedia Structural Integrity*, vol. 27, pp. 46-53, 2020.



- [25] N. Huda, and A. R. Prabowo, "Investigation of optimum ply angle using finite element (fe) approach: references for technical application on the composite navigational buoys," *Procedia Structural Integrity*, vol. 27, pp. 140-146, 2020.
- [26] D. Tsutsumi, D. Gyulai, A. Kovács, B. Tipary, Y. Ueno, Y. Nonaka, and K. Fujita., "Joint optimization of product tolerance design, process plan, and production plan in high-precision multi-product assembly," *Journal of Manufacturing Systems*, pp. 336-347, 2020.
- [27] N. D. Steenis, E. van Herpen, I. A. van der Lans, T. N. Ligthart, and H. C. M. van Trijp, "Consumer response to packaging design: The role of packaging materials and graphics in sustainability perceptions and product evaluations," *Journal of Cleaner Production*, vol. 162, pp. 286-298, 2017.
- [28] A. M. Oddershede, L. E. Quezada, J. E. Valenzuela, P. I. Palominos, and H. L. Ospina, "Formulation of a manufacturing strategy using the house of quality," *Procedia Manufacturing*, vol. 39, pp. 843-850, 2019.
- [29] I. N. Ismail, K. A. Halim, K. S. M. Sahari, A. Anuar, M. F. A. Jalal, F. Syaifoelida, and M. R. Eqwan, "Design and Development of Platform Deployment Arm (PDA) For boiler header inspection at thermal power plant by using the House of Quality (HOQ) Approach," *Procedia Computer Science*, vol. 105, pp. 296-303, 2017.
- [30] Z. Arifin, S. D. Prasetyo, S. Suyitno, D. D. D. P. Tjahjana, R. A. Rachmanto, W. E. Juwana, and T. Trismawati, "Rancang Bangun Alat Elliptical trainer Outdoor," *Mekanika: Majalah Ilmiah Mekanika*, vol. 19, pp. 104-112, 2020.

Copyright © 2021 by the authors. This is an open access article distributed under the Creative Commons Attribution License (CC BY-NC-ND 4.0), which permits use, distribution and reproduction in any medium, provided that the article is properly cited, the use is non-commercial and no modifications or adaptations are made.

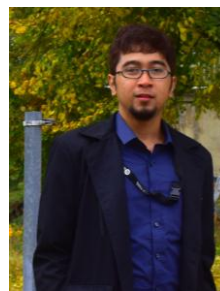


**Zainal Arifin** graduated bachelor's degree in Mechanical Engineering from Faculty of Engineering, Universitas Brawijaya. He was graduated master's degree in Mechanical Engineering from Faculty of Engineering, Universitas Gadjah Mada. He received his doctoral's degree in Mechanical Engineering from Universitas Brawijaya. He worked as a Associate Professor in the mechanical engineering study program at the Universitas Sebelas Maret. His works in the research of

photoanode manufacturing and nanofiber as important elements in solar cells. There are several research works funded by Ristekdikti and PNPB UNS. He is also active in publishing his research in conference and international journal including, JESTEC, Photoenergy Journal, Energies Journal, etc.



**Singgih D. Prasetyo** is a research student in Department of Mechanical Engineering, Faculty of Engineering, Universitas Sebelas Maret. He is active in organizations, now he is the head of the academic division of KMTM FT UNS. He is also an assistant lecturer in physics, production processes, and mechanical engineering. He is active in research and community service in the fields of solar cells, spray dryers, dyes, bio pore technology, organic compost, and biogas. The solar cell research was funded by the Student Creativity Program as the head researcher. He actives in the publication of engineering research results in international journals and conferences, including the Mechanical Engineering Journal, Mechanics Journal, ICOMERA, ICIMECE, ICESEAM, and others.



**Aditya R. Prabowo** is an Assistant Professor in Department of Mechanical Engineering, Universitas Sebelas Maret, Surakarta, Indonesia. He received Ph.D. degree and from Interdisciplinary Program of Marine Convergence Design, Pukyong National University, Busan, South Korea. Numerical simulation and design have become his interest in research. Currently, design development and mechanical analysis become his interest, which also involved him in the project of advanced metallic and non-metallic materials. He

serves as reviewer and editor in several publishers, including Elsevier, Springer Verlag, De Gruyter, MDPI AG and Hindawi. In the latest symposium – ICIMECE 2020, he acted as guest editor for Procedia Structural Integrity (Elsevier) and Open Engineering (De Gruyter).



**Joung H. Cho** is an Associate Professor and Lecturer in Interdisciplinary Program of Marine Convergence Design for industrial design. In same time, he acts as the head of a Korean scholarship program, BK21 plus MADEC in Pukyong National University, South Korea. He became a pioneer in several agreements and organizations in field of urban design, and a notable one is the agreement between Busan, South Korea and Surabaya, Indonesia in terms of spatial development. Besides design, he also paid his attention to research in marine design, such as structural capability against impact load, and attended several marine-related seminars and meetings.